



Institute of Aeronautics and Applied Mechanics

Finite element method 2 (FEM2)

Error estimation in a finite element

A - POSTERIORI ERROR APPROXIMATION

VECTOR OF THE AVERAGE STRESS AT NODE i :

$$\{\sigma\}_i^{ave} = \frac{\sum_{e=1}^k \{\sigma\}_i^e}{k}$$

where :

k - number of elements adjacent to node i

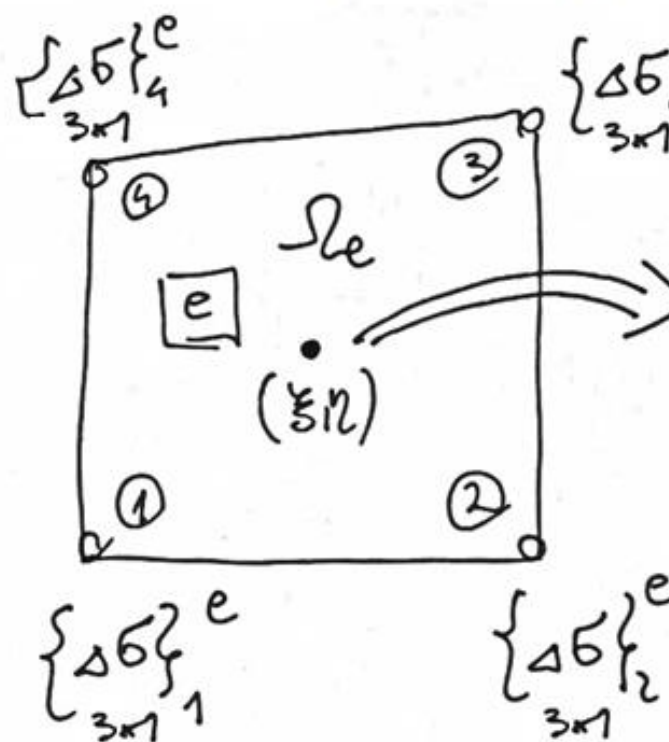
e - element number

$$\{\sigma\}_i^e = \begin{Bmatrix} \sigma_x \\ \sigma_y \\ \sigma_z \\ \tau_{xy} \\ \tau_{yz} \\ \tau_{zx} \end{Bmatrix}_i^e$$

THE STRESS ERROR IN ELEMENT e AT NODE i :

$$\underbrace{\{\Delta \sigma\}_i^e}_{6 \times 1} = \underbrace{\{\sigma\}_i^e}_{6 \times 1} - \{\sigma\}_i^{ave}$$

NODAL APPROXIMATION OF THE STRESS ERROR (QUAD 4 -NODE, PLANE STRESS)



$$\underbrace{\{\Delta \sigma(\xi, \eta)\}^e}_{3 \times 1} = \sum_{i=1}^4 \underbrace{\{\Delta \sigma\}_i^e}_{3 \times 1} \cdot N_i(\xi, \eta)$$

The maximum absolute value of any stress component :

$$\max \left(\left\{ \Delta \sigma \right\}_i^e \right)_{6 \times 1} \quad (\text{SDSG-IN ANSYS})$$

Strain energy in a finite element :

$$U_e = \frac{1}{2} \int_{\Omega_e} \underset{1 \times 6}{[B]}^e \cdot \underset{6 \times 1}{\{\epsilon\}}^e d\Omega_e = \frac{1}{2} \int_{\Omega_e} \underset{1 \times 6}{[B]}^e \underset{\substack{6 \times 6 \\ \uparrow \\ \text{Inverse constitutive matrix}}}{[D]}^{-1} \underset{6 \times 1}{\{\sigma\}}^e d\Omega_e$$

strain energy density in a finite element

$$u_e = \frac{U_e}{\int_{\Omega_e} d\Omega_e} \quad (\text{SEDELAS-IN ANSYS})$$

Total strain energy for the entire model:

$$U = \sum_{e=1}^{NOE} U_e$$

strain energy error in a finite element:

$$\Delta U_e = \frac{1}{2} \int_{\Omega_e} \underbrace{[\Delta \sigma]}_{1 \times 6}^T \underbrace{[D]}_{6 \times 6}^{-1} \cdot \underbrace{\{\Delta \sigma\}}_{6 \times 1}^e d\Omega_e \quad (\text{SEERR-IN ANSYS})$$

Total strain energy error in the entire model:

$$\Delta U = \sum_{e=1}^{NOE} \Delta U_e$$

The normalized energy error (for the entire model) is calculated as:

$$S = 100 \cdot \sqrt{\frac{\Delta U}{U + U}} \quad (\text{SEPC IN ANSYS})$$

The procedure of mesh refining:

$\Delta U_e = \text{const}$ for all FEs - "error equilibration"

$S < S_0$ for the entire model

S_0 - allowable value of S .

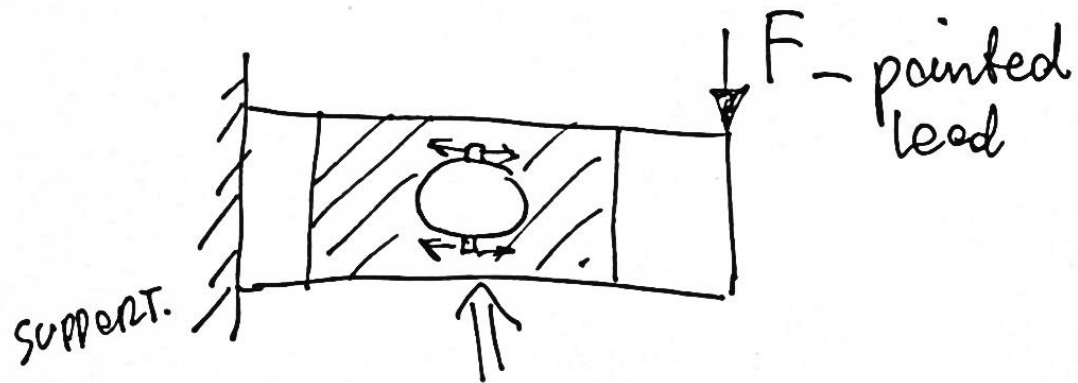
1°) Adaptive meshing - to correct mesh until:

$$\frac{\Delta U_e - \frac{\Delta U}{NOE}}{\frac{\Delta U}{NOE}} < \Delta U_0$$

↑ allowable value.

and $S < S_0$

2°) Selective adaptive meshing



TYPES OF REFINEMENT:

h - size decrease

p - polynomial order in shape functions

r - nodes rearrangement

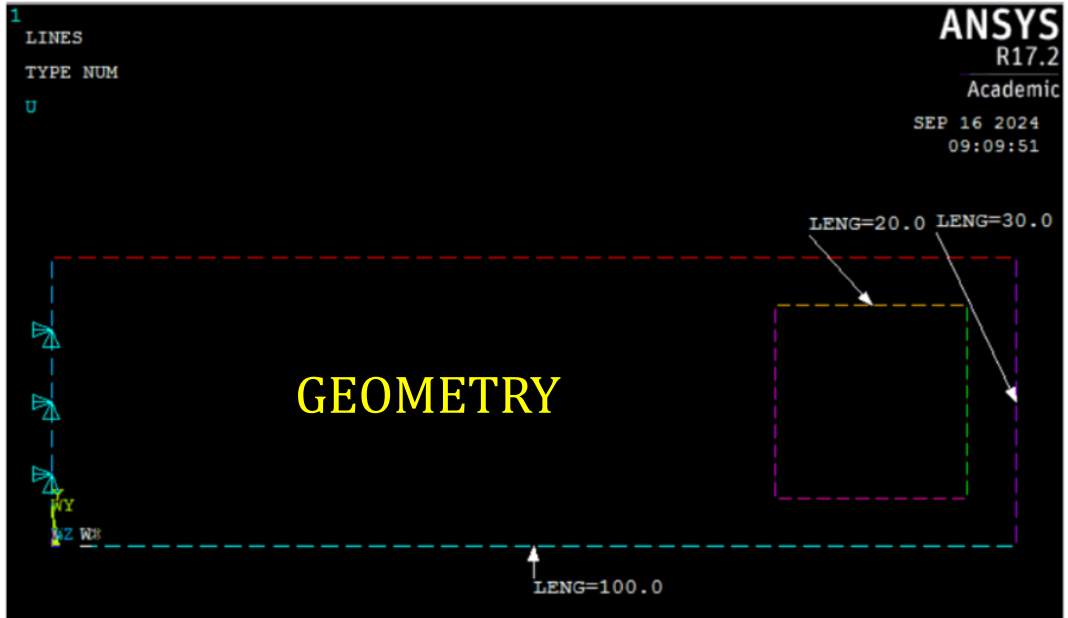
h_p - combination of " h " and " p "

Example. Plate

Type 1 PLANE182

Element technology K1

Enhanced Strain



WorkPlane Parameters



File

LIST SURFACE LOADS ON ALL SELECTED LINES

LINE	LOAD LABEL	VAL1	VAL2	VAL3
1	PRES	0.000	0.000	0.000
REFERENCE NUMBER FOR SLOPE COORDINATE SYSTEM= 0				
SLOPE DIRECTION= X				
COORDINATE LOCATION WHERE SLOPE IS ZERO= 0.00000000				
SLOPE VALUE (LOAD PER UNIT LENGTH)=-0.300000000				
2	PRES	0.000	0.000	0.000
REFERENCE NUMBER FOR SLOPE COORDINATE SYSTEM= 0				
SLOPE DIRECTION= Y				
COORDINATE LOCATION WHERE SLOPE IS ZERO= -45.00000000				
SLOPE VALUE (LOAD PER UNIT LENGTH)=-0.666666667				
3	PRES	0.000	0.000	0.000
REFERENCE NUMBER FOR SLOPE COORDINATE SYSTEM= 0				
SLOPE DIRECTION= X				
COORDINATE LOCATION WHERE SLOPE IS ZERO= 0.00000000				
SLOPE VALUE (LOAD PER UNIT LENGTH)=-0.300000000				

OWRGRPH

1

LINES

LINE NUM

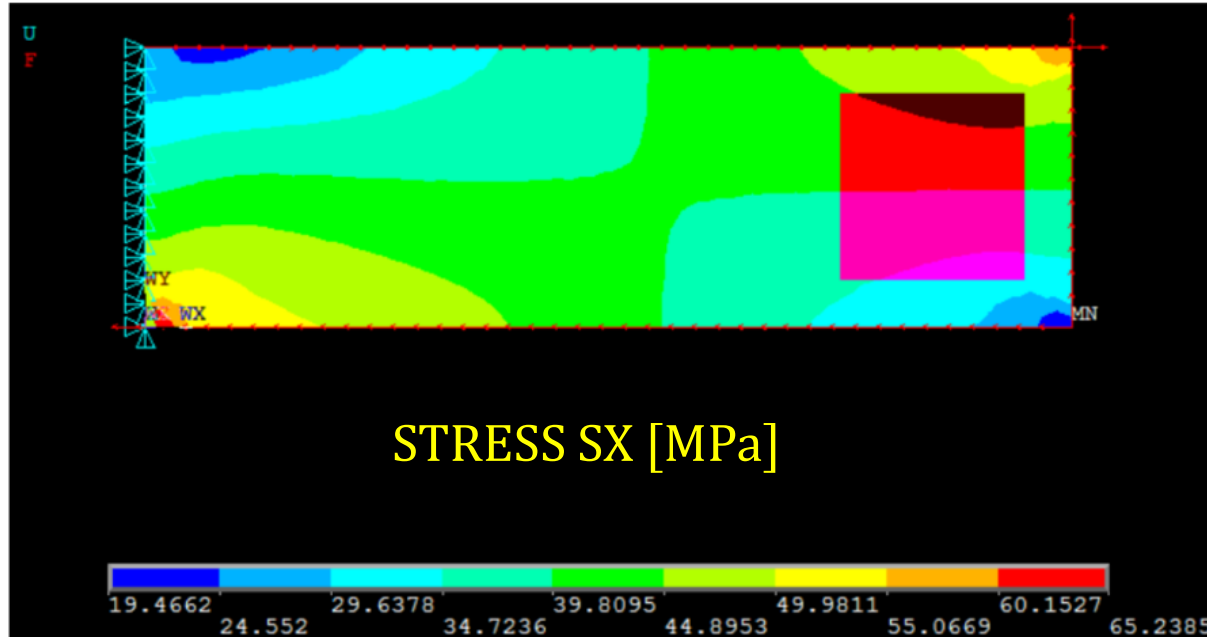
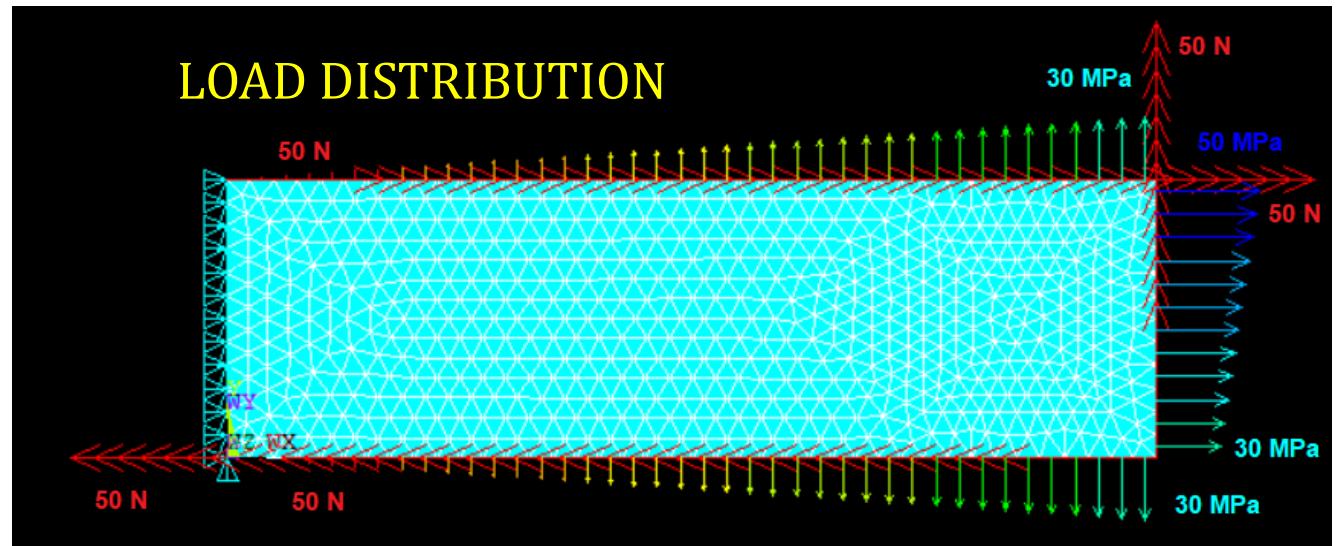
U

PRES-NORM

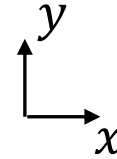
0

LOAD & BC

Example. Plate

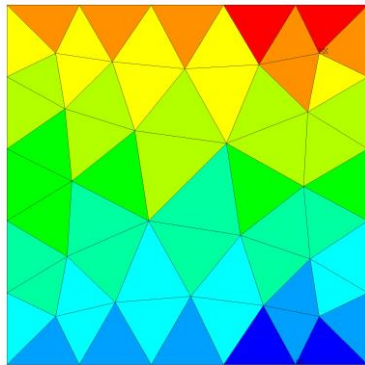


Normal stress in x direction (SX)



CST, 4mm

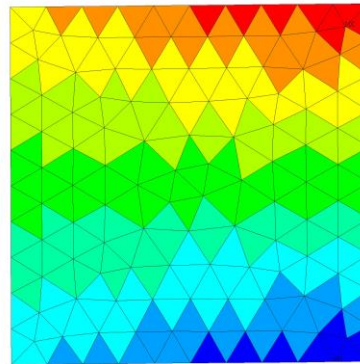
(free mesh)



SMN =29.9391
SMX =50.0735

CST, 2mm

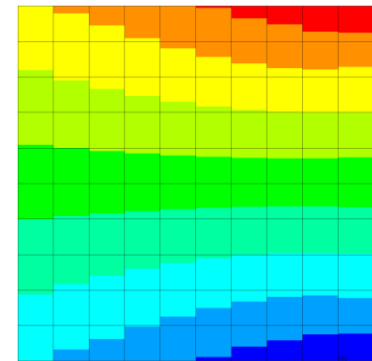
(free mesh)



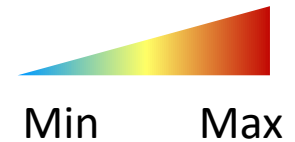
SMN =31.3547
SMX =48.657

QUAD-4node, 2mm

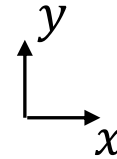
(mapped mesh)



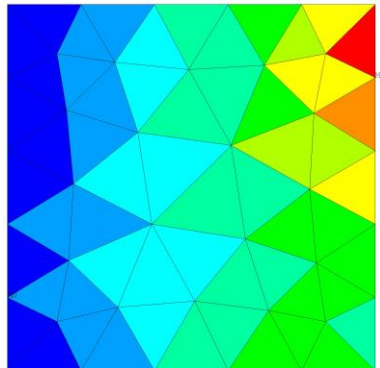
SMN =31.2497
SMX =48.819



Normal stress in y direction (SY)

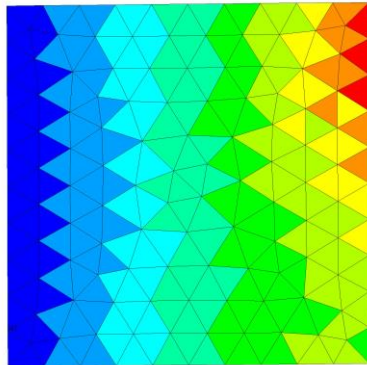


CST, 4mm



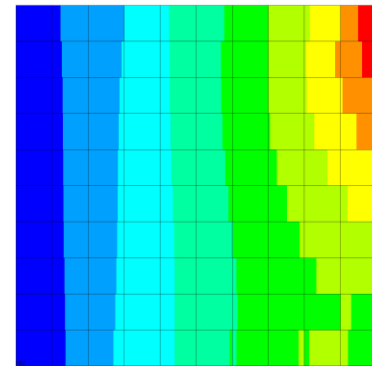
SMN =22.4161
SMX =31.2216

CST, 2mm



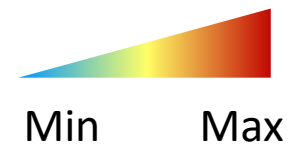
SMN =22.4077
SMX =30.4297

QUAD-4node, 2mm

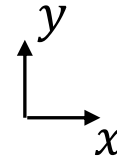


SMN =22.3724
SMX =30.5656

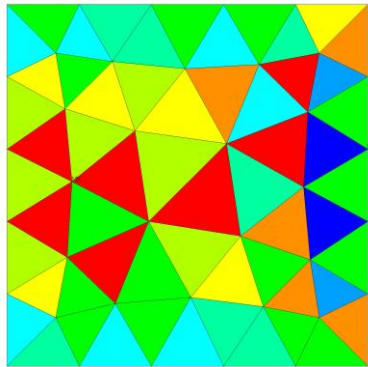
(MPa)



Shear stress (SXY)

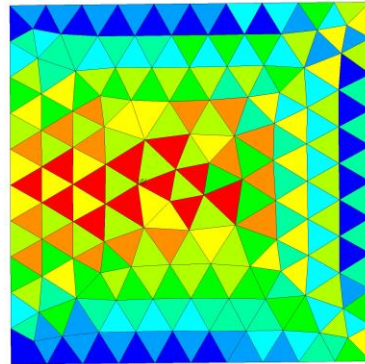


CST, 4mm



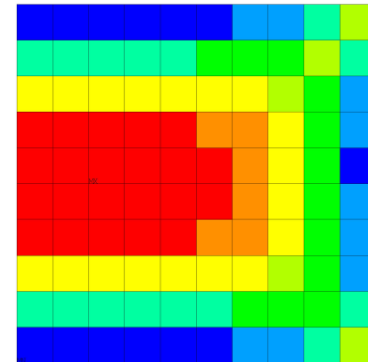
SMN =20.9712
SMX =22.9216

CST, 2mm



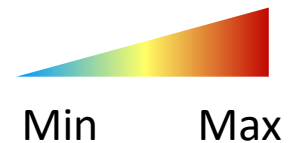
SMN =21.4276
SMX =22.7416

QUAD-4node, 2mm

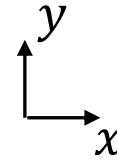


SMN =21.553
SMX =22.4885

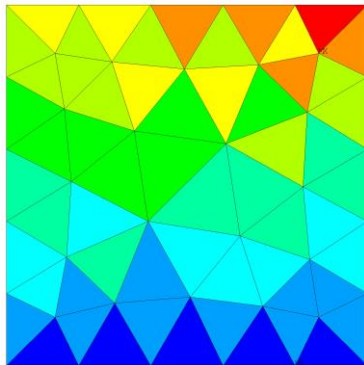
(MPa)



Von Mises stress (SEQV)

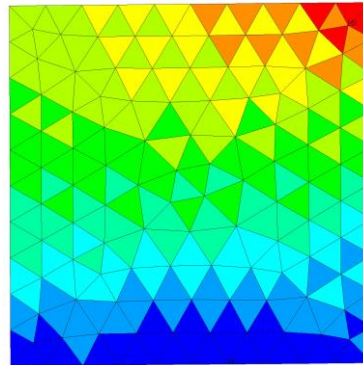


CST, 4mm



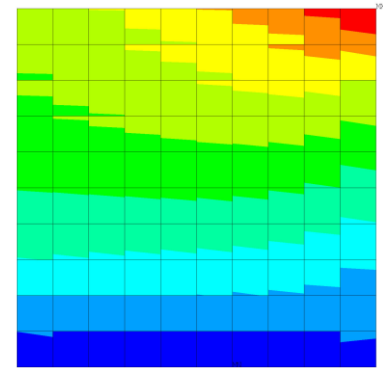
SMN =47.474
SMX =58.2418

CST, 2mm



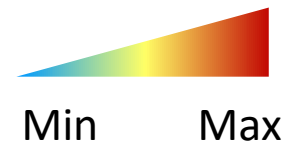
SMN =47.9007
SMX =57.2223

QUAD-4node, 2mm



SMN =47.9151
SMX =57.431

(MPa)



Elastic strain energy (SENE)

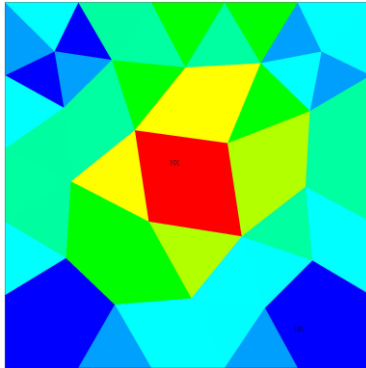
$$U_e = \frac{1}{2} \int_{\Omega_e} [\varepsilon] \{\sigma\} d\Omega_e \quad (\text{Nmm})$$

CST, 4mm

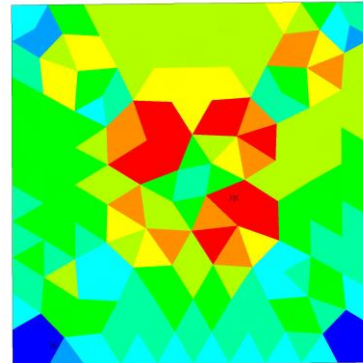
CST, 2mm

QUAD-4node, 2mm

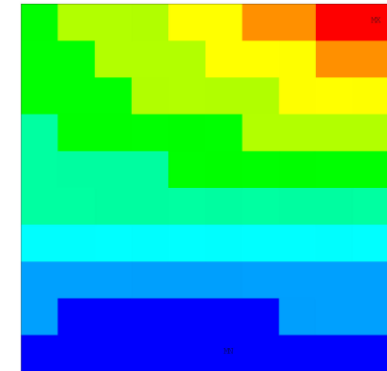
(SENE)



SMN = .029352
SMX = .093964

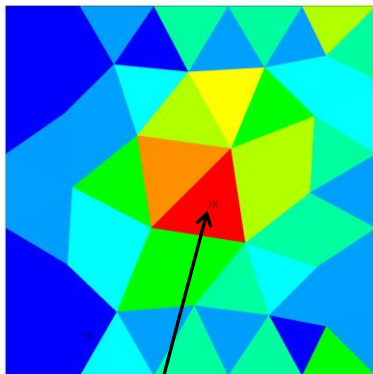


SMN = .007026
SMX = .019023

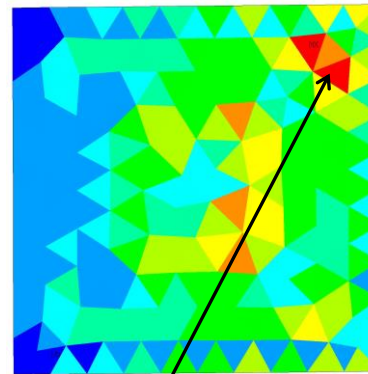


SMN = .024947
SMX = .035708

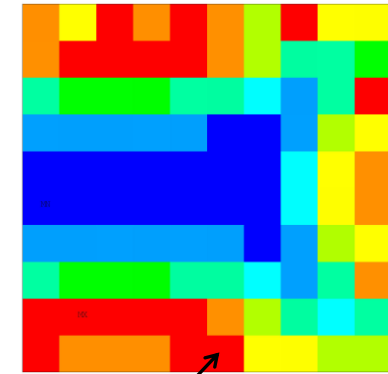
Error
(SERR)



SMN = .944E-05
SMX = .155E-03
(SEPC= 4.06%)



SMN = .434E-06
SMX = .624E-05
(SEPC= 1.92%)

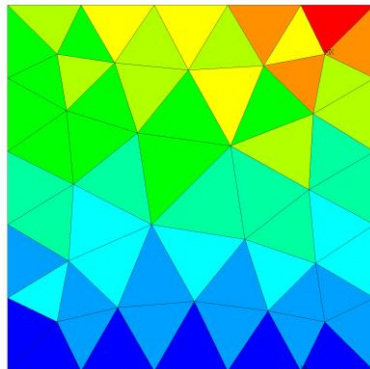


SMN = .309E-07
SMX = .945E-06
(SEPC= 0.51%)

Elastic strain energy density (SENDELAS)

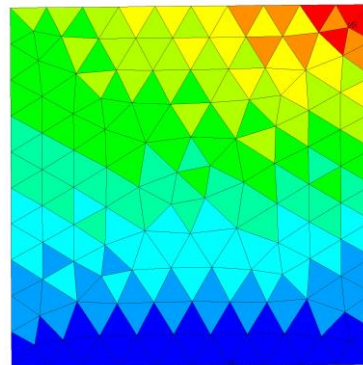
$$u_e = \frac{U_e}{\int_{\Omega_e} d\Omega_e}$$

CST, 4mm



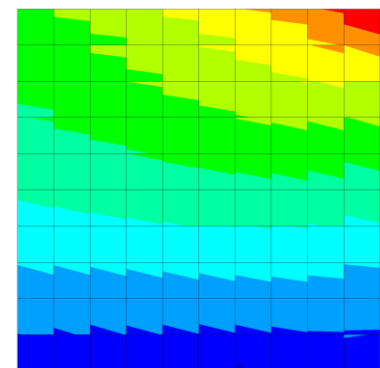
SMN = .005946
SMX = .009429

CST, 2mm



SMN = .006126
SMX = .009121

QUAD-4node, 2mm



SMN = .006113
SMX = .009243

(Nmm/mm³)

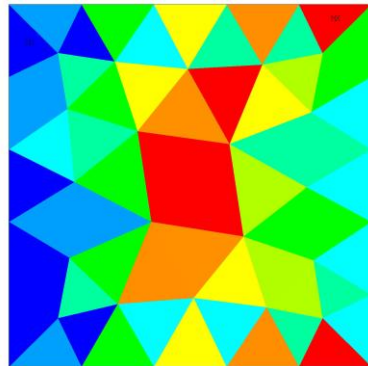


Min

Max

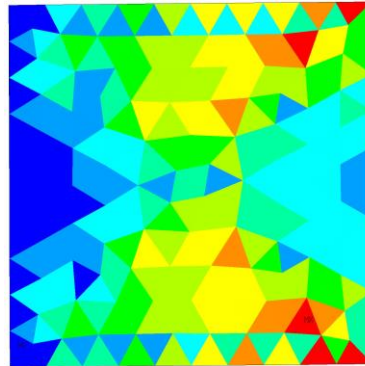
Maximum absolute value of error of any stress component (SDSG)

CST, 4mm



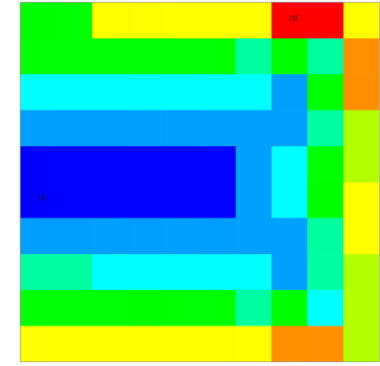
SMN =1.07305
SMX =3.37362

CST, 2mm



SMN =.490022
SMX =1.61018

QUAD-4node, 2mm



SMN =.045321
SMX =.338378

(MPa)

$$\max \left(\left\{ \Delta \sigma \right\}_i^e \right)_{6 \times 1}$$

